

SOLAR ROCKS

Thermal Analysis Presentation Part 1 Cold Fluid Hot fluid × Valve O Pump



2

SOLAR COLLECTOR (2 TO 1)



Possibility of ordering a small-scale parabolic trough. Awaiting return contact.

3

HEAT EXCHANGER (5 TO 3)



- Hot working fluid enters one side and heats the cold CO2 on the other.
- Shell and Tube design is preferred.
- Several sources for purchase are being considered.
- Estimate is around the \$800 range.

Figure 2.

THERMAL STORAGE ANALYSIS - OPTION 1

Lumped Capacitance approach:

Governing Equations:

$$\frac{T - T_{\infty}}{T_i - T_{\infty}} = e^{-\frac{hA_S t}{\rho V c}}$$

$$\tau = \frac{\rho V c}{h A_s}$$

$$Q = \rho V c \theta_i \Big[1 - e^{-\frac{t}{\tau}} \Big]$$

Т	Temperature of the Spheres	V	Volume of the chamber
Ti	Initial sphere temperature	с	Specific heat of the fluid
T∞	Temperature of the fluid in the chamber	t	Time that has passed
h	Heat transfer coefficient	τ	Thermal time constant
As	Surface are of the sphere	θi	Initial temperature Vs. Fluid Temperature
P	Density of the fluid		

Treat each sphere as an individual lump capacitance.

Break into sections of the chamber and numerically compute.



THERMAL STORAGE ANALYSIS- OPTION 2

Packed Bed Approach:

Governing Equations:

$$\frac{T_s - T_o}{T_s - T_i} = e^{-\frac{hA_{p,t}}{\rho V c_p A_{c,b}}}$$

$$\Delta T_{lm} = \frac{(T_{s} - T_{i}) - (T_{s} - T_{o})}{\ln(\frac{T_{s} - T_{i}}{T_{s} - T_{o}})}$$

$$q = hA_{p,t}\Delta T_{lm}$$

Ts	Steady state material temperature	V	Volume of the chamber
Ti	Inlet temperature	Ср	Specific heat of the fluid
То	Outlet temperature	Apt	total surface area of the packed bed
ΔTIm	Log mean temperature	Acb	cross sectional area of the packed bed
h	heat transfer coefficient	P	Density of the fluid

This approach assumes a steady packed bed temperature.

Errors in the code make this hard to determine if it is accurate.



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